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# **CFM03 Documentation**

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## 1.1 Problem1 - a

Describe the essential steps of the solution method. Include the discretized equations and implementation of boundary conditions.

In this problem set, we are supposed to solve the Navier-Stokes equations having continuity and momentum conservation equations together. Tensor forms of continuity and momentum equations are given below:

- Continuity (incompressible)

$$\frac{\partial u_i}{\partial x_i} = 0$$

- Momentum equation:

$$\frac{\partial u_i}{\partial t} + \frac{\partial u_i u_j}{\partial x_j} = -\frac{1}{\rho} \frac{\partial p}{\partial x_i} + \nu \frac{\partial}{\partial x_j} \left( \frac{\partial u_i}{\partial x_j} \right)$$

- Non-dimensionalization of the Navier-Stokes equations

In some cases, it is beneficial to non-dimensionalize the given transport equation because it eases the analysis of problem of interest, and also may reduce the number of parameters. The non-dimensionalized form of the Navier-Stokes equation can be achieved by first normalizing the primitive variables as followings:

$$\tilde{u}_i = \frac{u_i}{U_{\text{ref}}}, \quad \tilde{x}_i = \frac{x_i}{L_{\text{ref}}}, \quad \tilde{\rho} = \frac{\rho}{\rho_{\text{ref}}}, \quad \tilde{P} = \frac{P}{\rho_{\text{ref}} U_{\text{ref}}^2}, \quad \tilde{t} = \frac{t}{L/U_{\text{ref}}}$$

For the final form of non-dimensionalized Navier-Stokes equation, tilda,  $\tilde{\cdot}$ , will be dropped out for brevity and a new non-dimensional physical parameter  $Re$  that represents the flow inertia against the fluid viscosity is introduced. Now we got:

$$\frac{\partial u_i}{\partial t} + \frac{\partial u_i u_j}{\partial x_j} = -\frac{1}{\rho} \frac{\partial p}{\partial x_i} + \frac{1}{Re} \frac{\partial}{\partial x_j} \left( \frac{\partial u_i}{\partial x_j} \right)$$

where the Reynolds number is defined as:

$$Re = \frac{U_{\text{ref}} L_{\text{ref}}}{\nu}$$

- Artificial Compressibility Method (ACM)

In the artificial compressibility method (ACM), the continuity equation is modified adding an unsteady term with artificial compressibility  $\beta$ . To have this new form of continuity equation, an artificial equation of state that relates pressure,  $P$ , to artificial density  $\tilde{\rho}$  is employed as following form:

$$P = \frac{\tilde{\rho}}{\beta}$$

Finally, the modified continuity equation can then be recast as:

$$\frac{\partial P}{\partial t} + \frac{1}{\beta} \frac{\partial u_i}{\partial x_i} = 0$$

- Vector form of transport equations

Rewriting the previously derived non-dimensionalized continuity and momentum equation in vector form generates a simple format that eases implementation of the numerical method. The above transport equation can be newly formed as shown below:

$$\frac{\partial \vec{U}}{\partial t} + \frac{\partial \vec{E}}{\partial x} + \frac{\partial \vec{F}}{\partial y} = \frac{1}{\text{Re}} \left( \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) \vec{U}$$

where the each of vector elements are summarized below:

$$\vec{U} = \begin{bmatrix} P \\ u \\ v \end{bmatrix}, \quad \vec{E} = \begin{bmatrix} \frac{u}{\beta} \\ uu + P \\ uv \end{bmatrix}, \quad \vec{F} = \begin{bmatrix} \frac{v}{\beta} \\ uv \\ vv + P \end{bmatrix}$$

Now this is good to go further for discretization because the given task is to solve explicit form of discretization equation. Even though the derived form of transport equation is not linearized, each of vectors above are easily discretized in terms of their elements that are combinations of each primitive variables. Thus, in this project, actual discretization has been done from the driven transport equation above.

- Finding time step algorithm

In order to find time step that may stabilize the numerical solution, we need to know system convecting velocity as we pick the coefficient of spatial derivative terms in Burger's and Euler equations as the convection velocity. The driven system of equation is not a single partial differential equation but a set of three different partial differential equations. To find the convection speed of numerical information in the time and space domains, we need to first linearize the given system of equations and find the Eigen values. The linearization can be obtained by following process. The driven system of PDE should be reformulated in linearized set of equations:

$$\frac{\partial \vec{U}}{\partial t} + [A] \frac{\partial \vec{U}}{\partial x} + [B] \frac{\partial \vec{U}}{\partial y} = \frac{1}{\text{Re}} \left( \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) \vec{U}$$

Now we have found two coefficient matrices of convection terms and the spatial derivatives is now taken with respect to  $\vec{U}$  only. Despite the vector form, the PDE form is identical with Burger's equation. The coefficient matrices are below listed:

$$[A] = \begin{bmatrix} 0 & \frac{1}{\beta} & 0 \\ 1 & 2u & 0 \\ 0 & v & u \end{bmatrix}, \quad [B] = \begin{bmatrix} 0 & 0 & \frac{1}{\beta} \\ 0 & v & u \\ 1 & 0 & 2v \end{bmatrix}$$

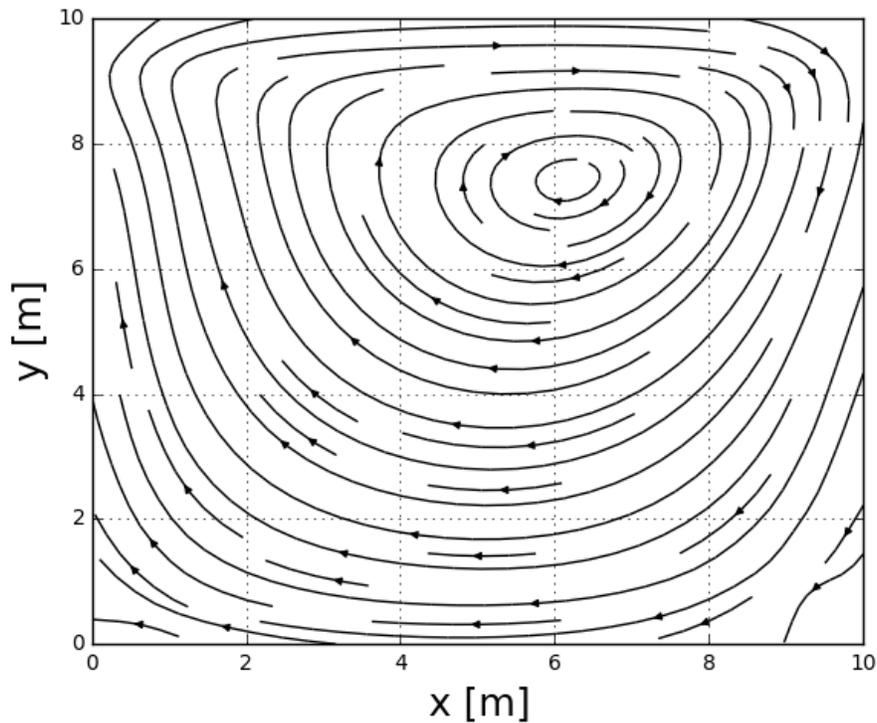
The resolved Eigen values of  $[A]$  and  $[B]$  matrices are  $u, u + a, u - a$  and  $v, v + a, v - a$ , respectively. Taking  $[A]$  for example, the maximum convection velocity that transmit the numerical information can then be  $|u| + a$ . Therefore, the Courant number for this case can also be determined by:

## 1.2 Problem1 - b, c

Consider the case when  $H = W$  (a square cavity). Here, the Reynolds number,  $Re = UW/\nu$ , characterizes the flow patterns. Compute the steady state solutions for both  $Re = 100$  and  $Re = 500$ . Plot the flow streamlines and centerline profiles ( $u$  vs.  $y$  and  $v$  vs.  $x$  through the center of the domain). For  $Re = 100$ , validate your method by comparing your results to data from given literature.

### 1.2.1 $Re = 100$

- $N_x N_y = 20 \times 20$



– u-velocity

– v-velocity

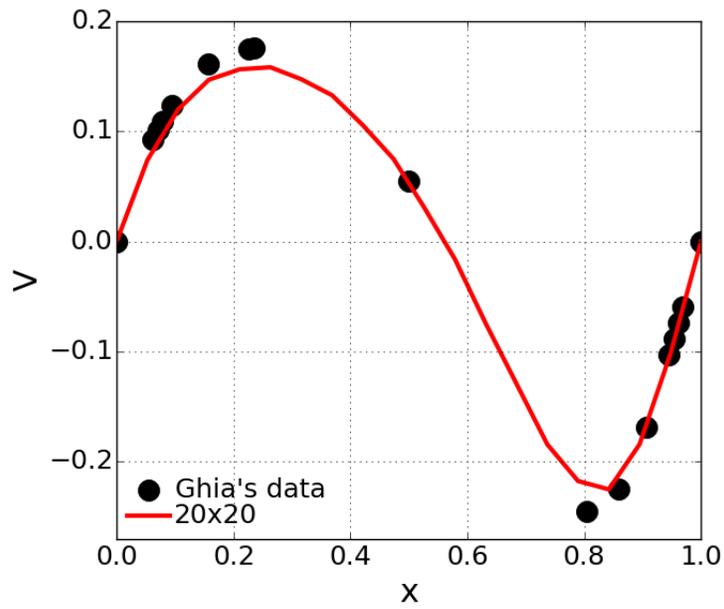
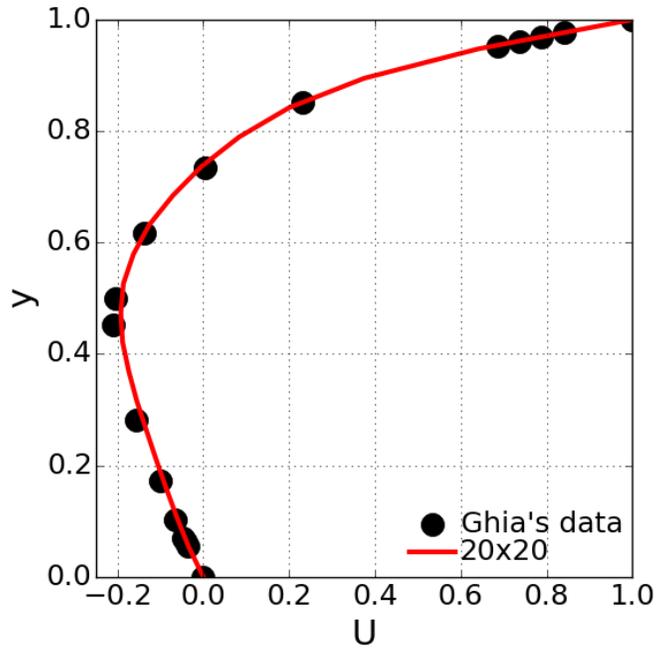
– **Observation**

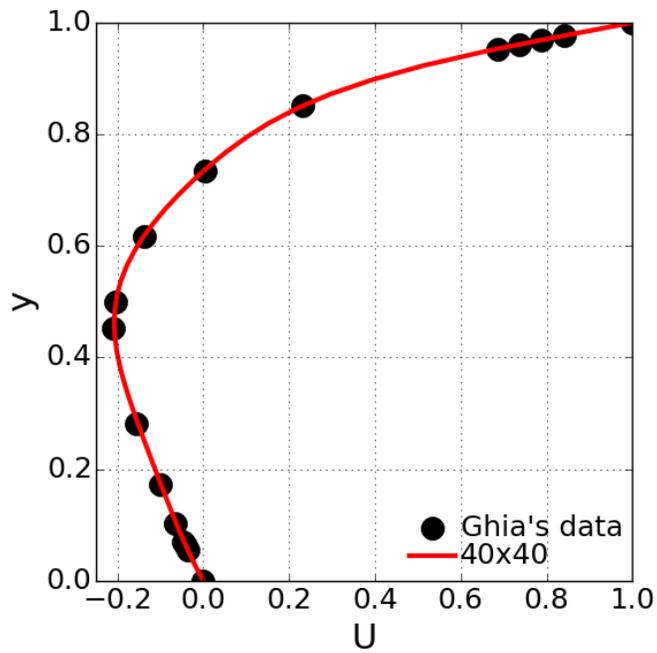
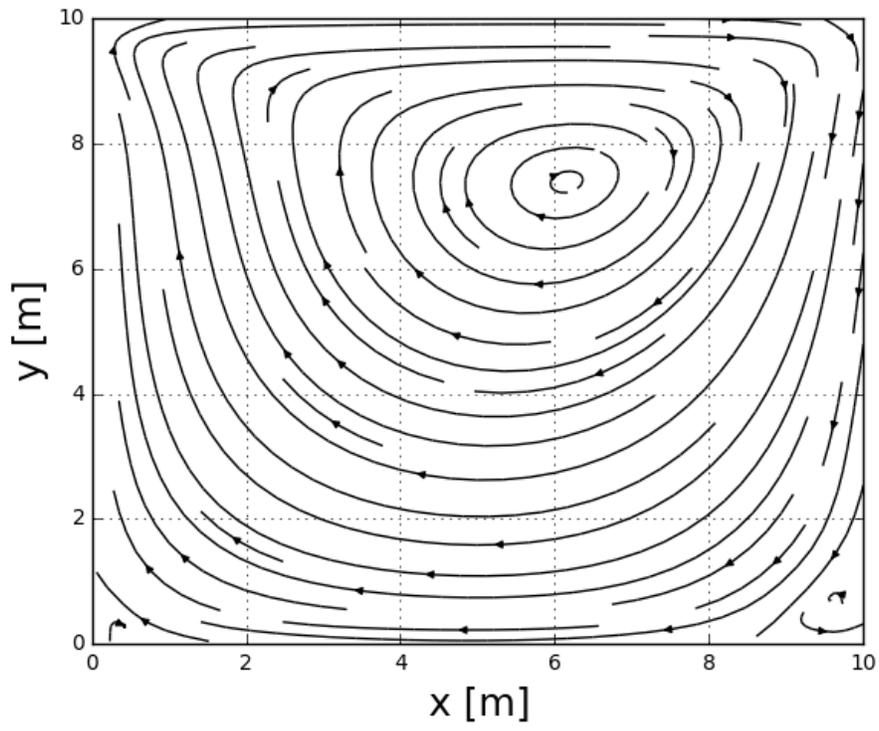
- \* Streamlines roughly forms and recirculation zone in the bottom right can be found.
- \* This coarse grid case shows bad estimation of  $u$  and  $v$ -velocity as compared to the Ghia's data

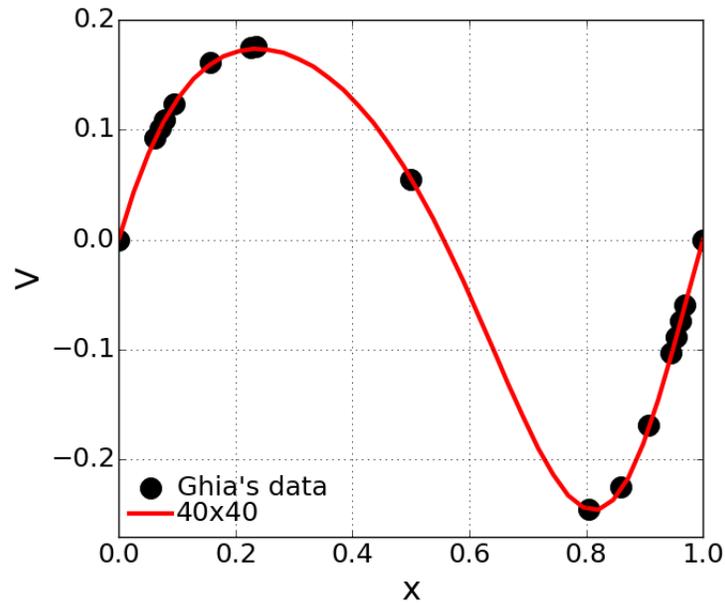
- $N_x N_y = 40 \times 40$

– u-velocity

– v-velocity







– **Observation**

- \* The predicted u- and v-velocity approached closer to the Ghia's data

•  $N \times N = 80 \times 80$

- u-velocity
- v-velocity

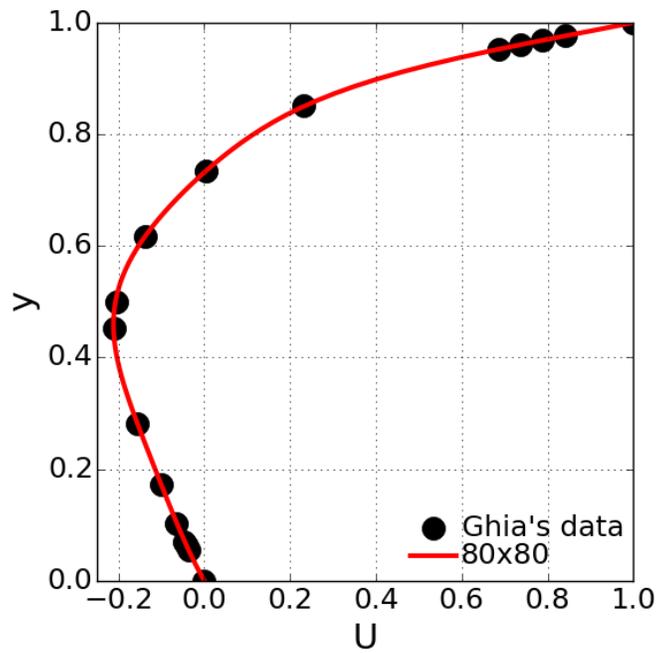
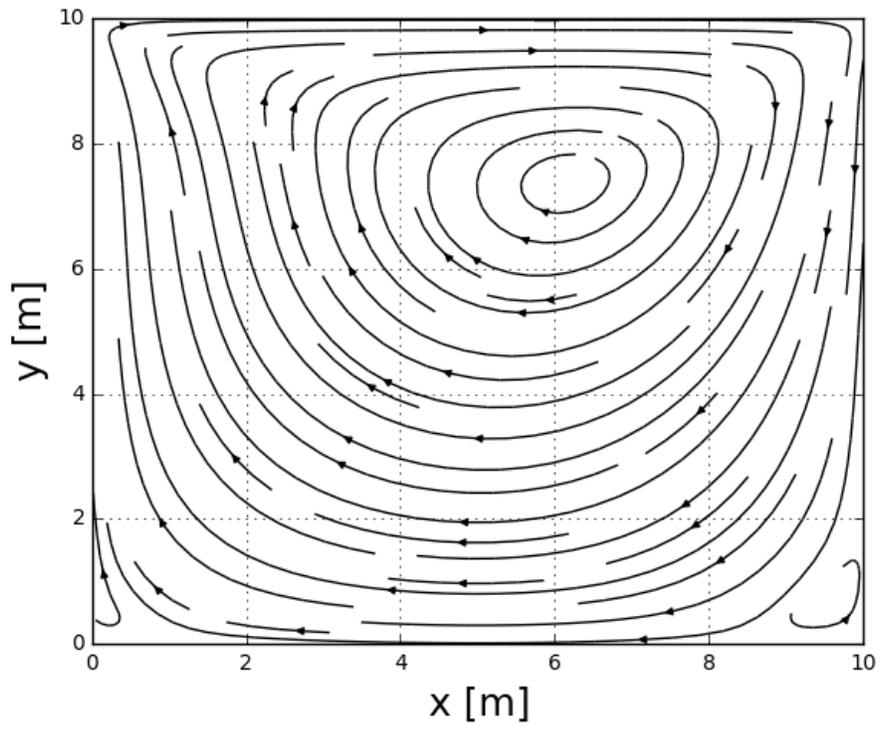
– **Observation**

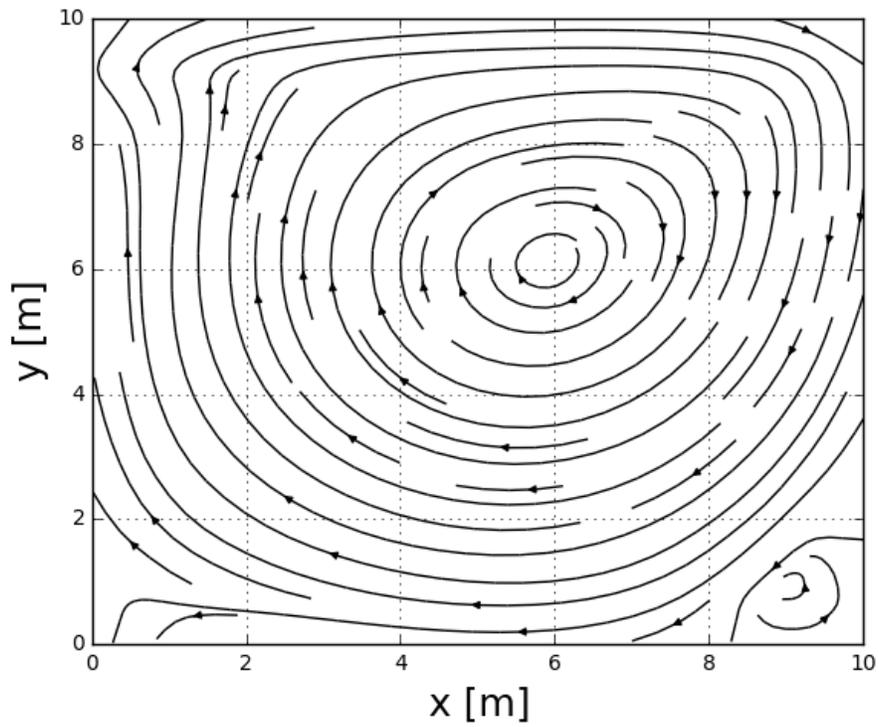
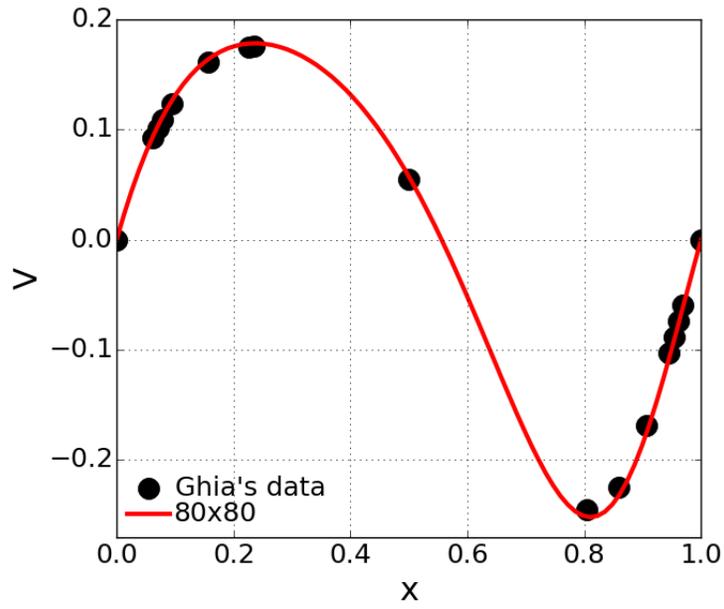
- \* The currently predicted data seems to be almost identical with the Ghia's solution.
- \* Recirculation zone in the bottom left and right seems more clear than the coarser grid cases.

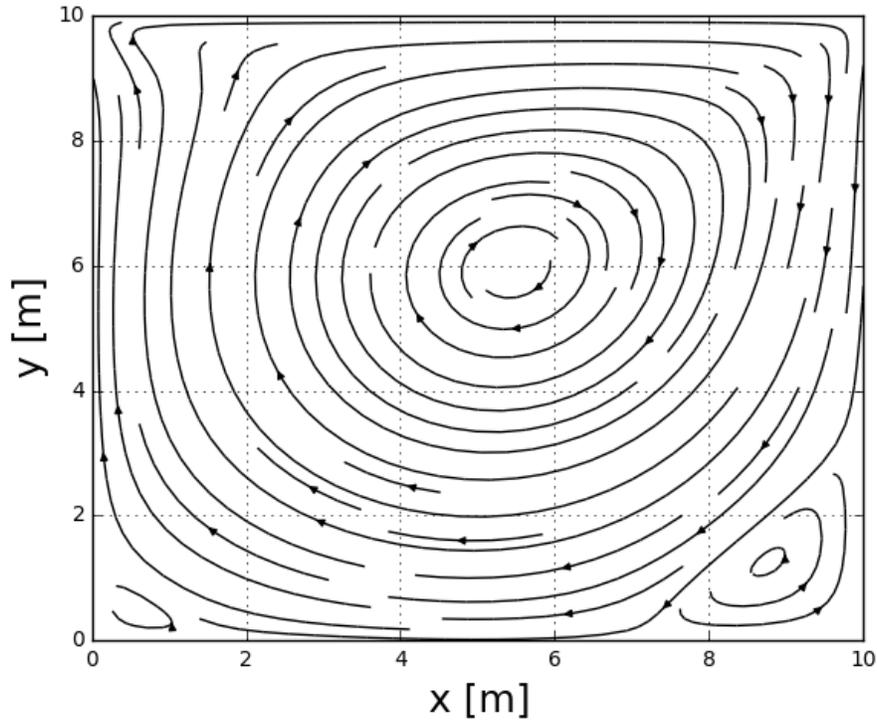
## 1.2.2 $Re = 500$

•  $N \times N = 20 \times 20$

•  $N \times N = 80 \times 80$







### 1.3 Problem1 - d

Examine the method stability with different grids. Determine the maximum time step that leads to a stable solution and compare it to the stability criteria.

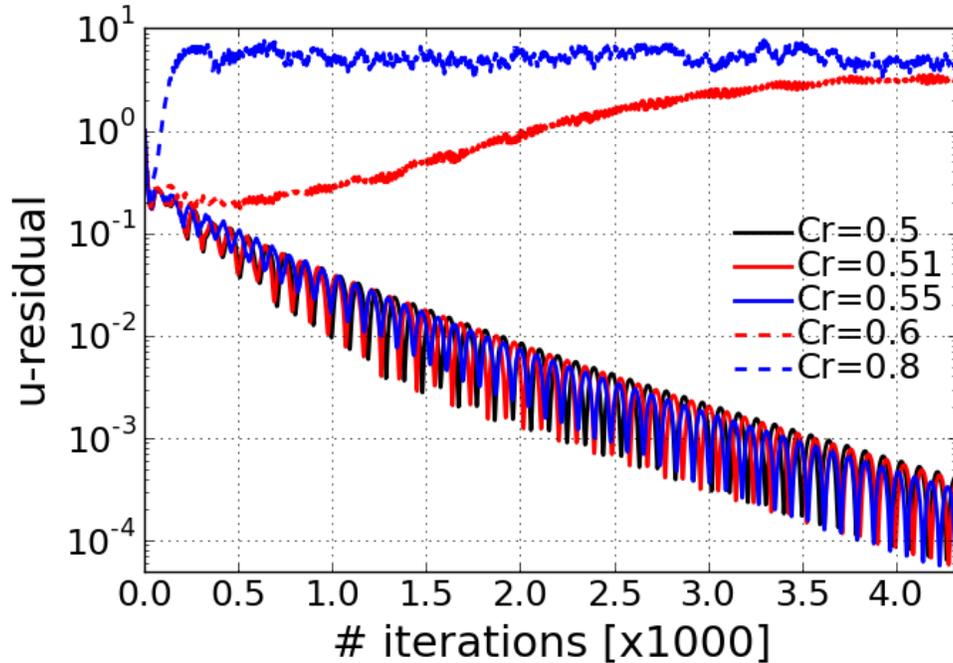
#### 1.3.1 Grid spacing test

Here, the stability test is performed with different set of grid spacing. To rule out other effect of numerical setup, *Courant* number and  $\beta$  remain constant whereas only grid spacing changes. Following table shows the stability check. *O* denotes the *stable* condition and *X* represents the *unstable* condition.

| NxN   | stability |
|-------|-----------|
| 10x10 | X         |
| 15x15 | X         |
| 16x16 | X         |
| 17x17 | O         |
| 18x18 | O         |
| 20x20 | O         |

#### 1.3.2 Maximum time step

In this code, the variable time step method is used to maintain stable numerically. Therefore, the code does not run with constant time step. The maximum time step test is performed with different set of *Courant* number condition. The grid spacing is fixed with 20x20 to have fast running of simulation.



| Courant # | dt at last iteration |
|-----------|----------------------|
| 0.5       | 0.007470             |
| 0.51      | 0.007620             |
| 0.55      | 0.008218             |
| 0.6       | 0.008653             |
| 0.8       | 0.007761             |

## 1.4 Problem1 - e, f

Compare your results with different grid resolutions to evaluate the numerical error and the order of the scheme.

### Re = 100

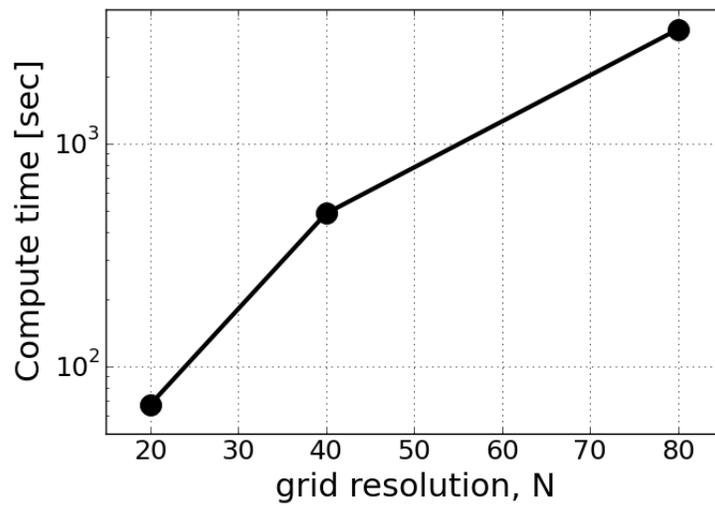
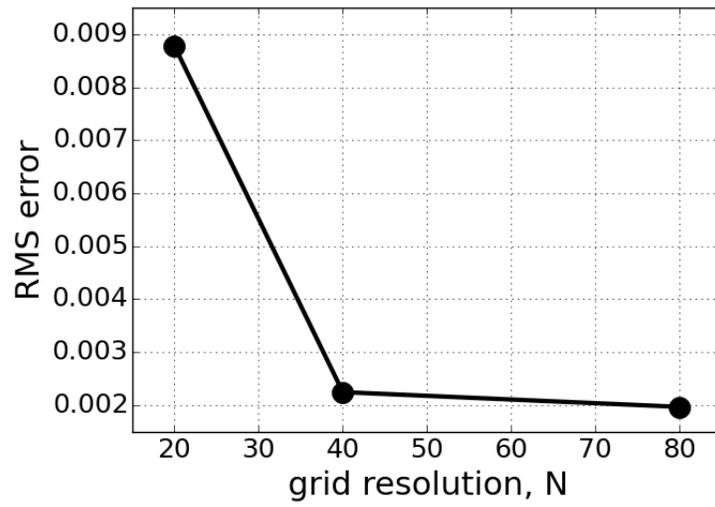
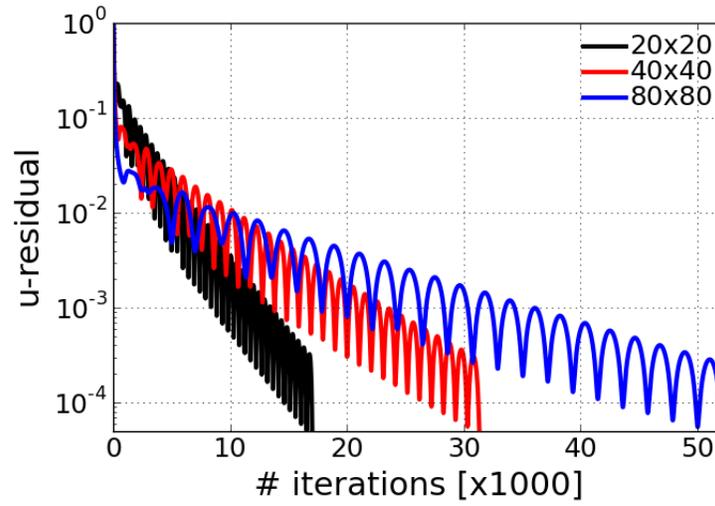
- Residual of u-velocity change in numerical iteration
- RMS error of u-velocity (reference: Ghia's u-velocity data)
- Computational time with different grid spacing

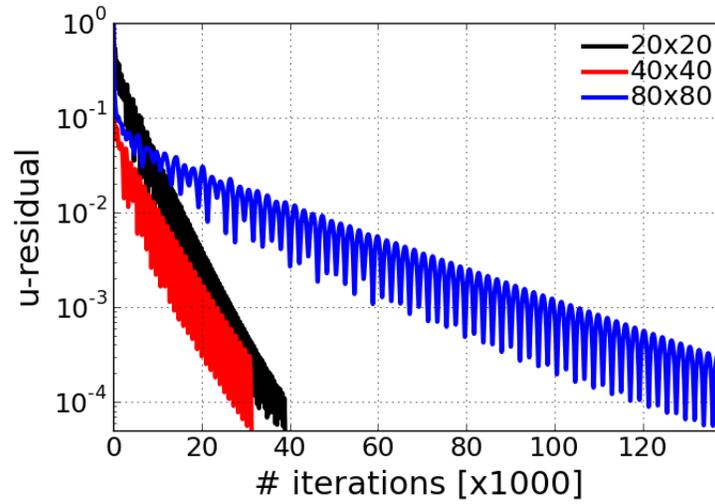
### Re = 500

- Residual of u-velocity change in numerical iteration

## 1.5 Problem1 - g

Repeat parts b and c for the case of  $H = 1.5W$  (except for validation). How does the flow change in this rectangular cavity as compared to the flow in a square cavity?





### 1.5.1 Re = 100

- $N \times N = 20 \times 30$
- $N \times N = 40 \times 60$
- $N \times N = 80 \times 120$

### 1.5.2 Re = 500

- $N \times N = 40 \times 60$

